

紧凑型压电式高性能快反镜结构设计

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摘要 针对便携式快反镜系统对小尺寸快反镜的需求,设计了一种紧凑型压电式驱动的快反镜结构。通过对快反镜的驱动元件、位移放大机构、解耦支撑的合理布局,具有 75 mm 通光口径的快反镜机械结构的外形尺寸为 90 mm×90 mm×33 mm。最后对快反镜进行了实验检测,结果显示紧凑型快反镜的角行程为 4.2 mrad,机械谐振频率为 671 Hz,表明该快反镜系统性能好,可满足小尺寸快反镜光学系统的应用需求。

关键词 光学器件; 快反镜; 紧凑型; 谐振频率; 自适应光学

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1 引言

快速反射镜(FSM)是一种能精确控制光束指向和具有稳定光束成像质量的精密光学装置,已经被广泛应用于自适应光学、图像稳定、精密跟踪、激光通讯及激光精密加工等领域^[1-7]。

近年来,国内外学者针对不同的应用领域和工作环境提出并研究了各种类型的 FSM。Janssen 等^[2]研制的 FSM 角行程为 10 mrad,谐振频率为 355 Hz,这款 FSM 可在低温环境下工作;Park 等^[7]为激光扫描系统研制的 FSM 的角行程为 2.8 mrad,谐振频率为 4 kHz;王恒坤等^[8]研制的动载体激光发射系统中 FSM 的角行程达 17.45 mrad 以上,指向精度优于 9.7 μrad;徐新行等^[9]研制的刚性支撑式 FSM 的角行程大于 2.91 mrad,谐振频率大于 100 Hz,这款 FSM 可在车载等工作环境恶劣的平台上工作;周子云等^[10]研制的 FSM 角行程达 17.45 mrad 以上,指向精度优于 4.6 μrad 以上。然而这些 FSM 的设计都未考虑体积,尤其是高度尺寸较大,不利于光学系统的小型化。一些光学系统,如激光计量、投影系统和星间光通信系统^[11-12]等,对

FSM 尺寸有着较为严格的要求,这类集成化光学系统对小体积的 FSM 系统有着迫切的应用需求。

本文设计了基于压电陶瓷致动器(PCSA)的紧凑型 FSM,利用杠杆放大机构增加了快反镜的角行程,采用了柔性解耦支撑结构消除两轴的耦合偏转,对 FSM 进行了工作原理分析,并对其偏转角度和谐振频率等重要特性进行了输出方程推导。实验结果表明,该快反镜系统性能优良,可满足集成化光学系统的需求,对于工程实践具有重要意义。

2 紧凑型快反镜结构设计

2.1 快反镜工作原理

常用的 FSM 致动器有音圈电机和 PCSA 两种。音圈电机是一种基于洛伦兹力的直驱电机,尺寸一般比 PCSA 大,不利于 FSM 的小型化设计。PCSA 是一种利用压电体的逆压电效应形成驱动或力输出的精密定位装置,较之音圈电机,具有结构简单、体积小、可控性好、响应速度快、输出力大、换能效率高等优点。如图 1(a)所示,根据小型 FSM 设计要求所制备的 PCSA 几何尺寸为 14 mm×14 mm×17 mm,位移行程 δ_p 为 21 μm(即±10.5 μm),如图 1(b)所示。

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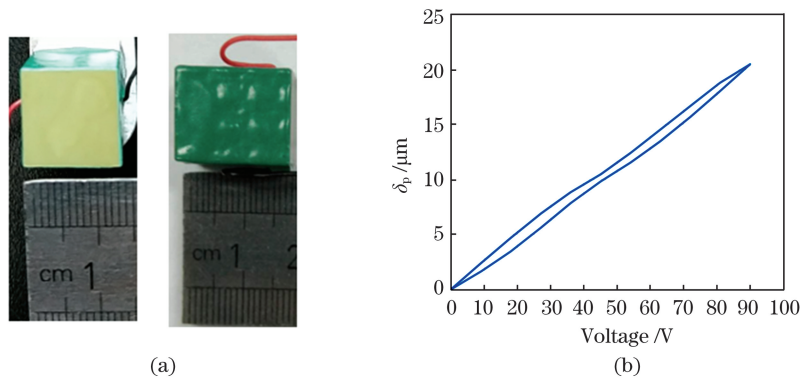


图 1 用于紧凑型快反镜的 PCSA。(a)PCSA 实物;(b)PCSA 的位移特性

Fig. 1 PCSA for the compact fast steering mirror. (a)Photograph of the PCSA;(b)stroke response of PCSA

由于制备的 PCSA 行程较小,在 FSM 结构中设置了位移放大机构。FSM 由反射镜面、解耦柔性支撑、基于柔性铰链的杠杆位移放大机构、PCSA、球头垫片等组成。4 个 PCSA 正交分布在镜面的偏转轴上,如图 2(a)所示。以镜面绕 Y 轴偏转为例,FSM 的工作原理如图 2(b)所示:镜面部分固定在解耦柔性支撑(铰链 B 和铰链 B')之间,铰链 A、铰链 B、位于两铰链之间的刚体杆 AB 组成的杠杆放大机构与 Y 轴对称的铰链 A'、铰链 B'、杆 A'B'工作原理完全相同。当 PCSA 发生微位移 δ_p ,铰链 B 会

发生更大的位移 δ ,可实现对 PCSA 微位移的放大。当处于 X 轴的一对 PCSA 分别发生大小相同方向相反的微位移,此时杆 AB 和杆 A'B'会有方向相反的偏转角。两个刚体杆的偏转会带动镜面产生更大的偏转角度,镜面的偏转角方向与两刚体杆的偏转方向相反,通过控制位于 X 轴的 PCSA 的驱动电压即可控制反射镜面在该方向的偏转角。镜面绕 X 轴偏转时,工作原理与绕 Y 轴偏转类似,FSM 的解耦柔性支撑铰链与 PCSA 处于平行位置,这样设计使得 FSM 结构更紧凑、体积更小,如图 2(c)所示。

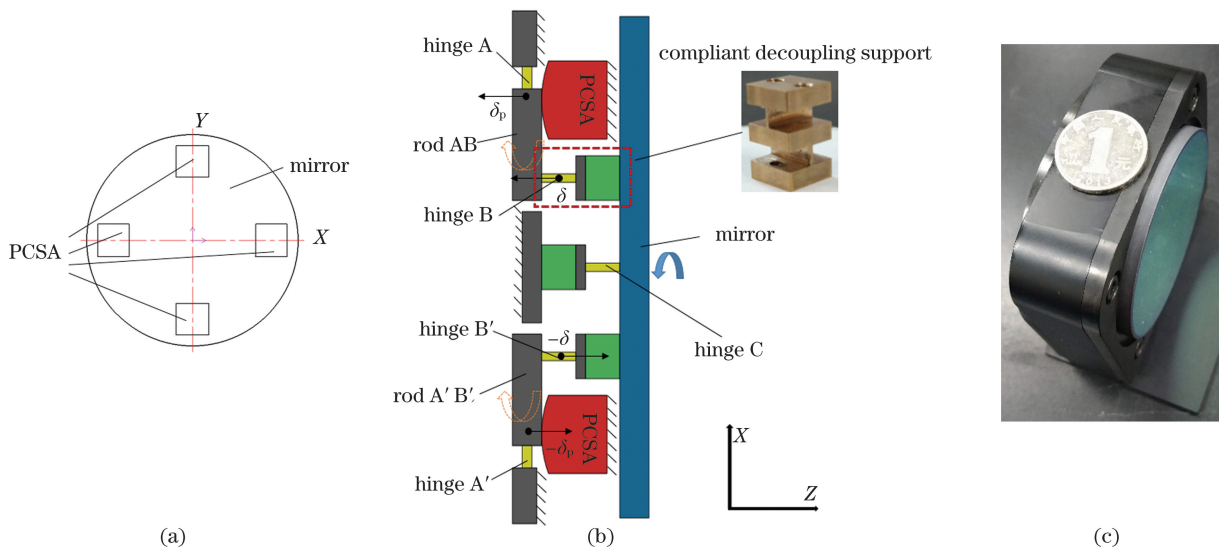


图 2 紧凑型 FSM 结构示意图。(a) PCSA 的分布示意;(b)紧凑型 FSM 镜面绕 Y 轴偏转工作原理示意;(c)紧凑型 FSM 实物照片

Fig. 2 Structure diagram of compacted FSM. (a) Distribution of PCSA; (b) working principle of compacted FSM deflecting around Y axis; (c) photograph of the compacted FSM

2.2 偏转角度分析

分析偏转角度时同样以镜面绕 Y 轴偏转为例,当位于 X 轴的一对 PCSA 在驱动电压作用下发生微位移 δ_p ,如图 3 所示,铰链 B 的位移 δ 为

$$\delta = \frac{d_1 + d_2}{d_1} \delta_p \quad (1)$$

考虑到镜面偏转角度较小,偏转角度 θ 可表示为

$$\theta = \frac{\delta}{d_3} \quad (2)$$

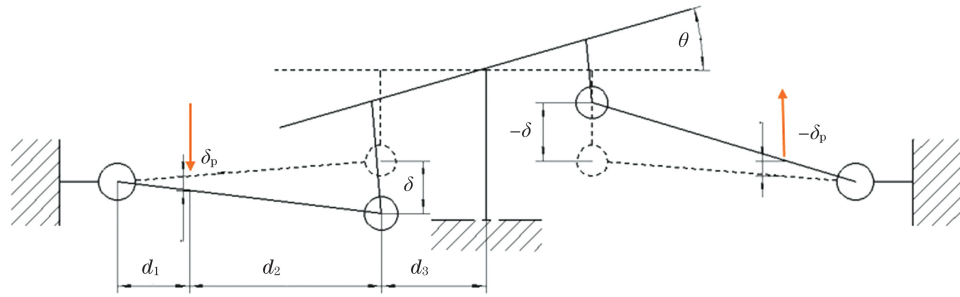


图 3 紧凑型快反镜的静态分析模型

Fig. 3 Static lumped stiffness model of the compacted FSM

联立(1)和(2)式,可得到偏转角度 θ 与 PCSA 位移 δ_p 的关系为

$$\theta = \frac{d_1 + d_2}{d_1 d_3} \delta_p \quad (3)$$

快反镜的几何尺寸参数如表 1 所示,代入(3)式可求得快反镜的偏转角行程为 ± 2.357 mrad。

表 1 快反镜的几何尺寸参数

Table 1 Geometrical parameters of the FSM

Parameter	Value
d_1/mm	7
d_2/mm	15
d_3/mm	14

2.3 谐振频率分析

FSM 的谐振频率决定了系统带宽和工作时的响应速度,是 FSM 非常重要的性能指标。由于偏转台和杆 AB 的谐振频率很高(超过 3 kHz),因此可视为刚体。PCSA 通过球形垫片与 FSM 其他部件刚性接触,会为整个结构提供附加刚度,因此 PCSA 可简化为刚度为 K_p 、当量质量为 m_p 的弹簧质量系统。紧凑型 FSM 的动态分析模型如图 4 所示,根据镜面的运动状态,广义模态坐标定义为镜面的偏转角度 θ 。刚体杆 AB 和 A'B' 的偏转角 α 可表示为

$$\alpha = \frac{\delta_p}{d_1} \quad (4)$$

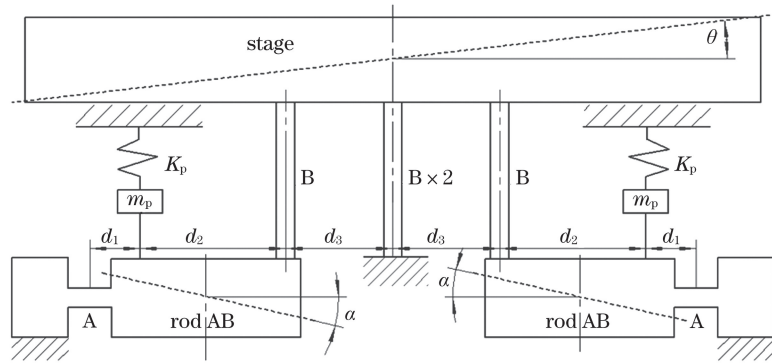


图 4 紧凑型快反镜动态分析模型

Fig. 4 Dynamic analysis model of the compact FSM

根据(1)~(4)式,可得镜面偏转角 θ 与 α 的关系为

$$\alpha = \frac{d_3}{d_1 + d_2} \theta \quad (5)$$

根据拉格朗日方程,得

$$\frac{d}{dt} \left(\frac{\partial \Delta E}{\partial \dot{\theta}} \right) - \left(\frac{\partial \Delta E}{\partial \theta} \right) = 0, \quad (6)$$

$$\Delta E = E_k - E_p, \quad (7)$$

式中: $\dot{\theta}$ 为镜面偏转角度的导数。 E_k 和 E_p 分别为系统的动能和势能,可分别表达为

$$E_k = \frac{1}{2} I_m (\theta')^2 + I_{AB} (G\theta')^2 + m_p (d_1 G\theta')^2, \quad (8)$$

$$E_p = K_A (G\theta)^2 + 2(1 + G^2 + 2G) K_B \theta^2 + K_p (d_1 G\theta)^2, \quad (9)$$

式中: I_m 和 I_{AB} 分别为镜面部分和刚体杆 AB 的转动惯量; K_A 和 K_B 分别为柔性铰链 A 和柔性铰链 B 的转动刚度; K_p 是 PCSA 为整个系统提供的附加刚度;常数 $G = \frac{d_3}{d_1 + d_2}$ 。系统的运动微分方程为

$$M\theta'' + K\theta = 0, \quad (10)$$

式中: $M = I_m + 2I_{AB}G^2 + 2m_p(d_1G)^2$, $K = 2GK_A + 4(1+G^2+2G)K_B + 2K_p(d_1G)^2$ 。

由此可得到系统的谐振频率为

$$f = \frac{1}{2\pi} \sqrt{\frac{K}{M}} \quad (11)$$

由(11)式可知,紧凑型 FSM 的谐振频率和系统的刚度(K_A, K_B, K_p)呈正相关,与系统的负载(I_{AB}, I_m, m_p)呈负相关。根据快反镜各部件参数(如表 2 所示),快反镜空载时的谐振频率为 1723.8 Hz,装配单晶硅反射镜面后,谐振频率为 707 Hz,可应用于大多数光学系统。

表 2 快反镜各部件参数

Table 2 Parameters of components of the FSM

Parameter	Value
Stiffness of hinge $K_A / (\text{N} \cdot \text{m} \cdot \text{rad}^{-1})$	90
Stiffness of hinge $K_B / (\text{N} \cdot \text{m} \cdot \text{rad}^{-1})$	18
Stiffness of PCSA $K_p / (\text{N} \cdot \text{m}^{-1})$	2.1×10^7
Rotary inertia of rod AB $I_{AB} / (\text{kg} \cdot \text{m}^2)$	8.16×10^{-6}
m_p / kg	0.046
Rotary inertia of stage with mirror $I_m / (\text{kg} \cdot \text{m}^2)$	5.37×10^{-5}
Rotary inertia of stage without mirror $I'_m / (\text{kg} \cdot \text{m}^2)$	2.02×10^{-6}

3 实验测试

为了对紧凑型 FSM 的工作原理和特性分析结果进行验证,进行了角行程和谐振频率测试。如图 5 所示,所用实验仪器包括高压放大器、动态自准直仪和示波器。其中高压放大器驱动 PCSA 发生位移,使 FSM 镜面产生偏转,动态自准直仪检测镜

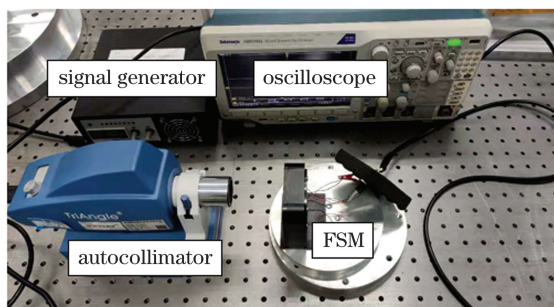


图 5 紧凑 FSM 实验测试系统实物

Fig. 5 Photograph of compacted FSM experimental measurement system

面的偏转角度,而示波器采集高压放大器的输出电压和动态自准直仪的信号。

3.1 角行程

图 6 为在频率为 2 Hz、驱动电压幅值为 90 V 下测得的偏转角度-时间曲线,测试结果显示紧凑型 FSM 的两个方向角行程均为 4.2 mrad。实验与理论有 12.23% 的误差,造成误差的主要原因是理论模型中柔性铰链的变形被认为是纯粹的转动,而实际有微小的横向位移。

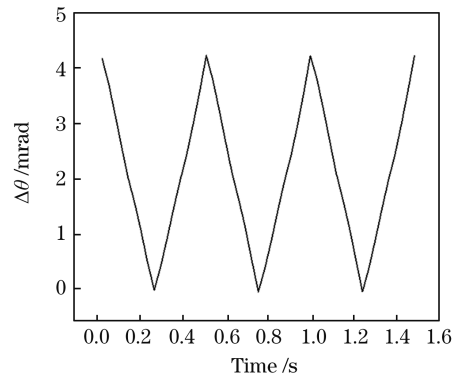


图 6 快反镜的偏转角度-时间曲线

Fig. 6 Relationship between the tilt angle and the time of the FSM

3.2 谐振频率

在 FSM 装配镜面前,对压电陶瓷施加正弦扫频信号,可得到 FSM 空载谐振频率。如图 7(a)和(b)所示,空载 FSM 沿 X 方向和 Y 方向的谐振频率分别为 1700 Hz 和 1692 Hz。在装配直径为 75 mm,厚度为 13 mm 的单晶硅反射镜面后,测得 X 方向和 Y 方向谐振频率分别为 671 Hz 和 676 Hz,如图 7(c)和(d)所示。空载 FSM 和负载后的 FSM 的理论谐振频率与实际谐振频率的误差分别为 1.8% 和 5.36%,造成误差的主要原因是加工误差和 PCSA 的刚度误差。

4 结 论

提出了一种紧凑型压电式快反镜,通过合理布局杠杆放大机构和解耦柔性支撑,减小了快反镜的体积。研究了所设计快反镜的工作原理,并对其进行了偏转角度和谐振频率的分析。实验结果表明,紧凑型快反镜的角行程为 4.2 mrad,谐振频率为 671 Hz,性能与 PI 公司同口径 S-340 型产品接近,且体积更小,有着良好的工程应用价值。

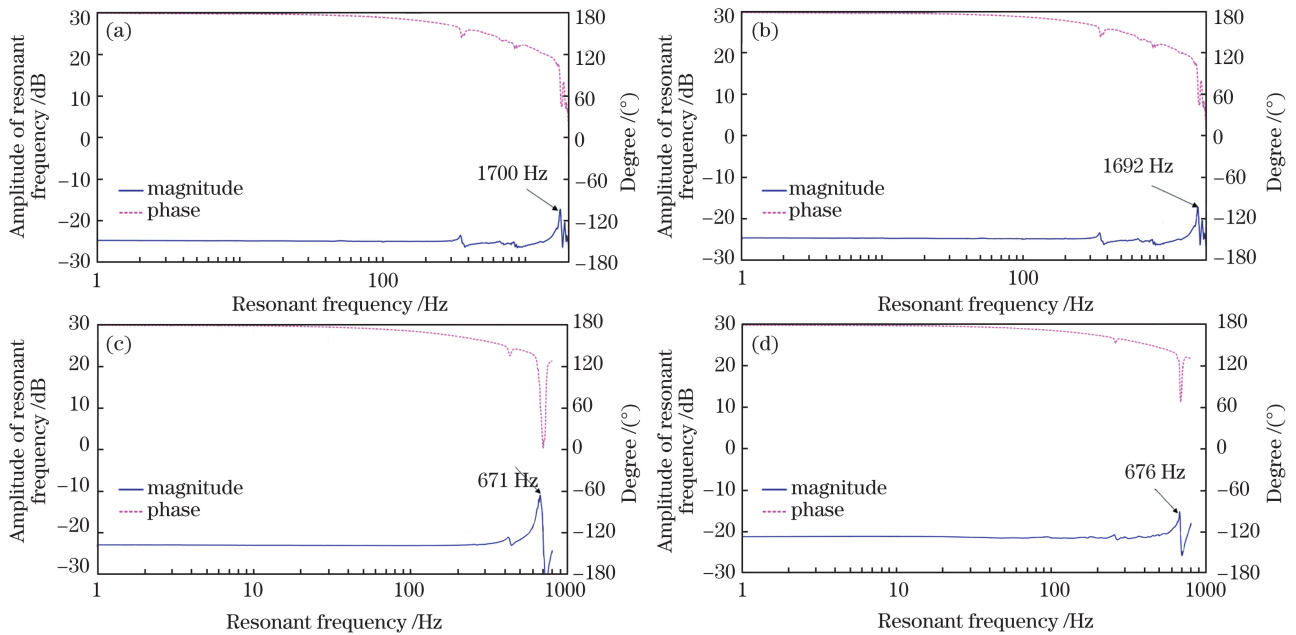


图 7 紧凑型 FSM 谐振频率检测结果。(a) 空载, X 方向检测结果; (b) 空载, Y 方向检测结果; (c) 装配镜面后, X 方向检测结果; (d) 装配镜面后, Y 方向检测结果

Fig. 7 Resonance frequency test results of the compact FSM. (a) Result along X direction without mirror; (b) result along Y direction without mirror; (c) result along X direction with mirror; (d) result along Y direction with mirror

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Structure Design of Compact Piezoelectric Fast Steering Mirror with High Performance

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Abstract

Objective The developments in adaptive optics, image stabilization, fast laser scanning, and optical communications have resulted in the increasing demand for high-performance fast steering mirrors (FSMs). In some optical systems, such as laser metrology, projection system, and inter-satellite optical communication, FSM size is crucial. In integrated optical systems, there is an urgent application demand for portable FSM systems. However, the existing FSM design does not consider the volume, especially height size, which is not conducive to miniaturization of optical systems. To meet the requirements of a portable FSM system for small-size FSMs, a compact piezoelectric driven FSM structure was designed. We expect the FSM not only to be portable but also to have excellent performance.

Methods There are two types of FSM driving actuators: voice coil actuator and piezoelectric ceramic stack actuator (PCSA). Voice coil actuator is a direct drive motor based on Lorentz force, its size is generally larger than PCSA, and its environmental adaptability is also poor. For the miniaturization of FSM, we chose a PCSA with a height of only 17 mm. Because the displacement of the PCSA is small, we set a lever displacement amplification mechanism in the FSM structure. In the structural design of FSM, a decoupled flexure hinge was parallel to the PCSA. Through a reasonable layout of driving elements, displacement amplification mechanism, and decoupling support of the mirror, the overall dimension of the FSM, especially the height dimension, can be significantly reduced. We established a theoretical model for FSM and analyzed its angular stroke and resonance frequency. The theoretical results show that the angular stroke and resonance frequency of FSM reach 4.714 mrad and 707 Hz, respectively, which are consistent with our expected performance.

Results and Discussions The overall dimension of the mechanical structure of 75-mm diameter transparent mirror is 90 mm × 90 mm × 33 mm. We tested the angular stroke and resonant frequency of the FSM. The experiment results show that the angular stroke of the compact FSM is 4.2 mrad (Fig. 6). There is an error of 12.23% between the experiment and theory results; the main reason for this error is that the deformation of the flexure hinge in the theoretical model is considered a pure rotation, but there is a small lateral displacement in practice. The mechanical resonance frequency is 671 Hz (Fig. 7), the error between the theoretical value and the measured value of resonance frequency is 5.36%. The main causes of the errors are machining errors and PCSA stiffness errors. The experiment results show that the FSM has a good performance and can meet the application requirements of a portable FSM system.

Conclusions We design a compact FSM that is portable and has high performance based on the application requirements of portable FSM for small-size and integrated optical systems. We choose PCSA with small stroke and height and use a one-stage amplification mechanism in the FSM structure. PCSAs and decoupling support hinges are in parallel position. This design is the first example in FSM with a lever amplification mechanism. Through reasonable structure design, the size of FSM is significantly reduced. In this article, a theoretical model for FSM is established, and the angular stroke and resonant frequency of FSM are derived. The experiment results are consistent with the theoretical results. Finally, based on the experiment, the established FSM is not only portable but also exhibits excellent performance. Compared with the same caliber product S-340 of PI company, the compact FSM has similar performance, smaller volume, and good engineering application value.

Key words optical devices; fast steering mirror; compact; resonant frequency; adaptive optics

OCIS codes 230.0250; 260.5740; 230.4040